



TOMSK POLYTECHNIC UNIVERSITY

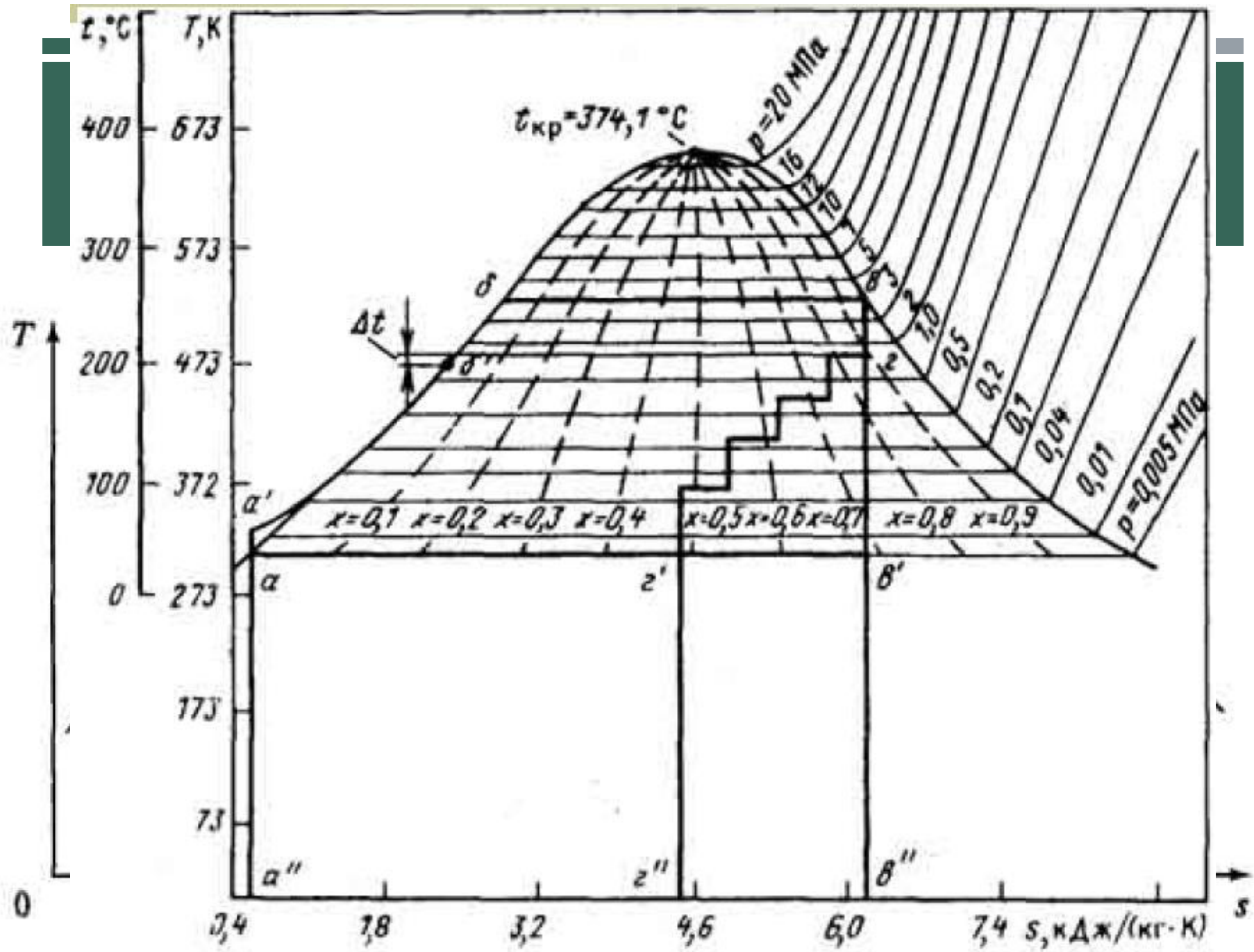
REGENERATIVE HEATING OF THE FEED WATER (RHFV)

MAIN FEATURES

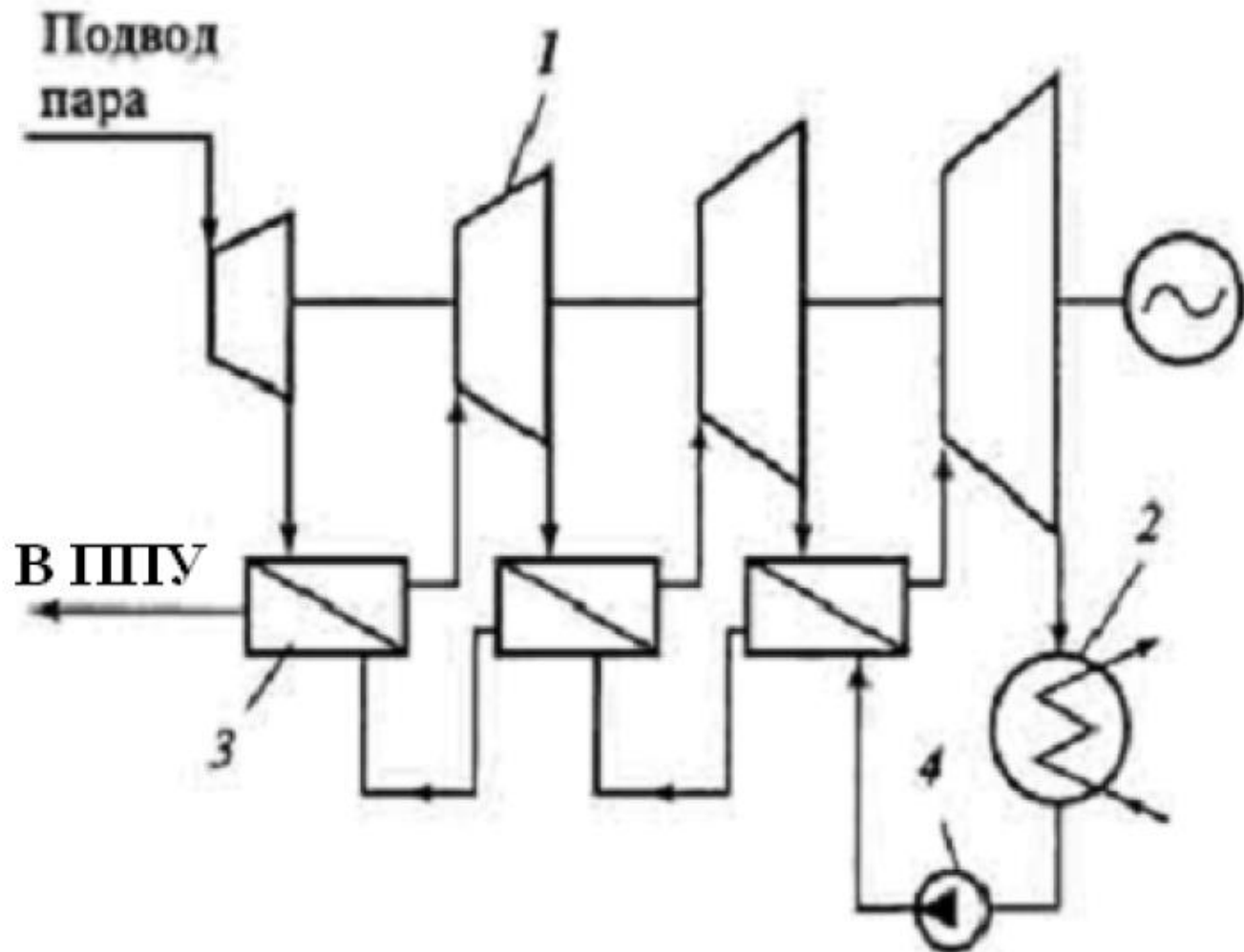
- Theoretical cycle with heat regeneration
- Real cycle with heat regeneration
- Effect of RHF_W on the efficiency of NPP
- Distribution of regenerative bleed into turbine
- Steam and drainage cooling

EFFECT OF FEED WATER TEMPERATURE ON EFFICIENCY

- To improve efficiency of NPP the average temperature of heat supplied to cycle should be increased. When ability to increase temperature on the inlet of steam turbine is depleted this could be reached by increasing the temperature of the feed water on the inlet of steam generator.
- The most efficient method to increase temperature on the inlet of steam generator is application of regenerative heating of feed water.



SCHEME OF THEORETICAL CYCLE WITH REGENERATIVE HEATING



FINITE THEORETICAL CYCLE WITH REGENERATIVE HEATING

Finite theoretical regenerative cycle:

- Infinite amount of regenerative bleeds;
- Infinitively low heat drops at each stage.

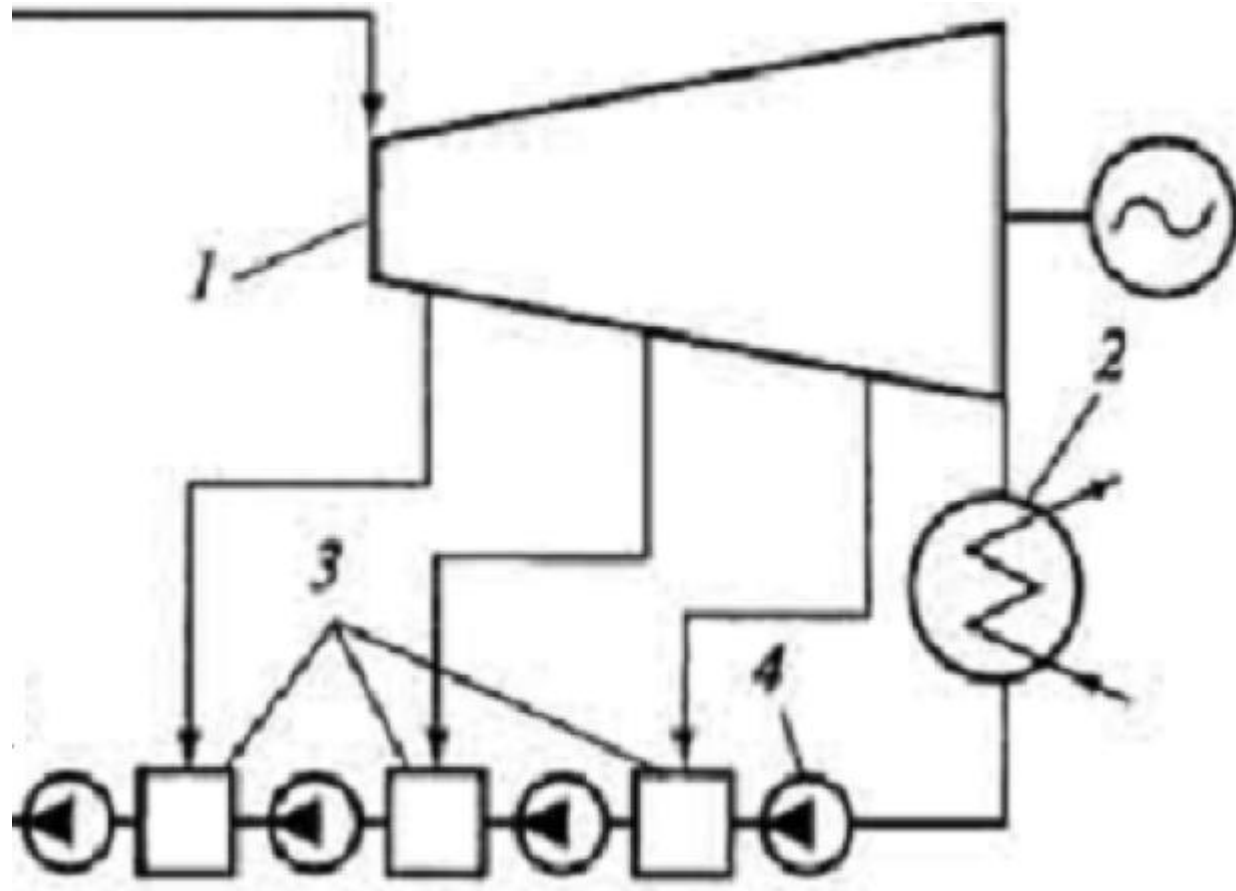
Impossibility of theoretical regenerative cycle:

- Unacceptable humidity in the end of expansion process;
- Large dimensions of heaters, bleed cross-section area and pipelines;
- High pressure losses.

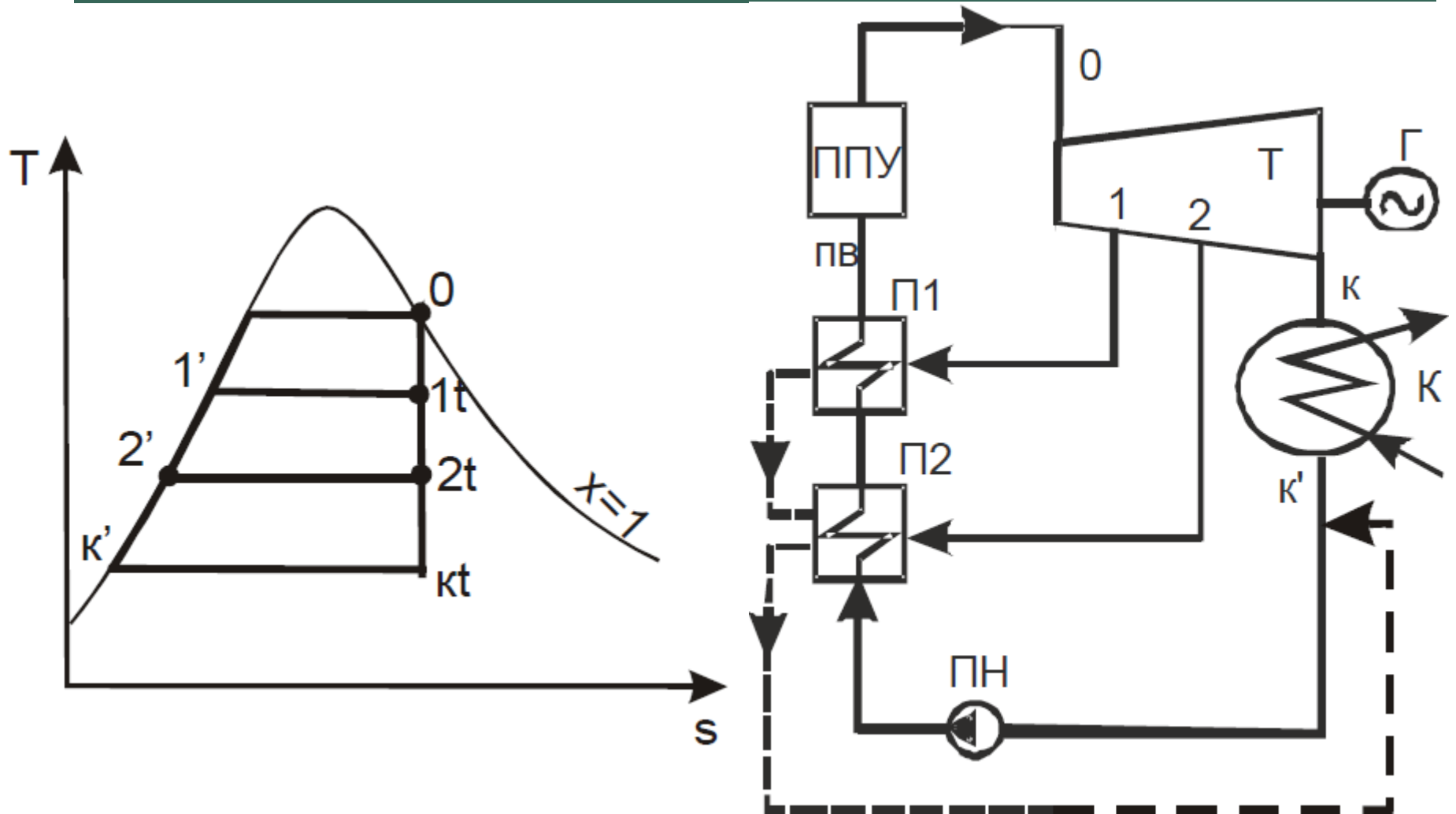
THREE STAGE RHFWS OF THE OPEN TYPE

One stage of RHFWS includes:

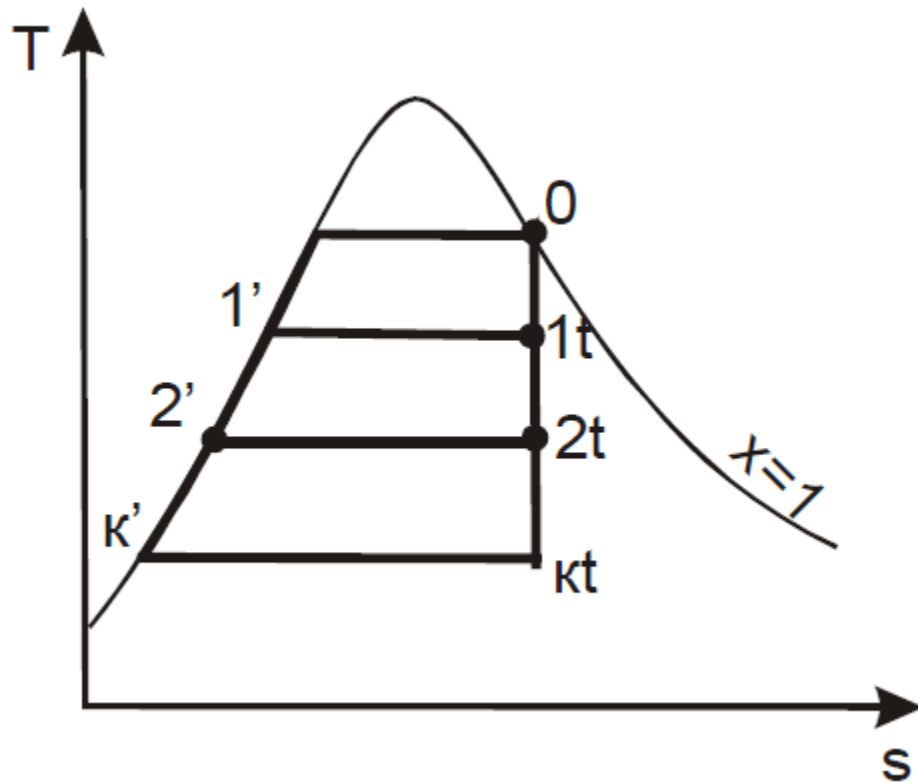
- Steam bleed;
- Heater;
- Steam pipeline;
- Pump (optional);
- Drain line (optional).



SCHEME AND CYCLE OF TWO STAGE RHFV OF THE CLOSED TYPE



WORK OF TURBINE IN SCHEME WITH TWO STAGE RHFWS



Theoretical work:

$$l_t = l_I + l_{II} + l_k = (h_0 - h_{1t}) + (1 - \alpha_1)(h_{1t} - h_{2t}) + (1 - \alpha_1 - \alpha_2)(h_{2t} - h_{kt}) = h_0 - \alpha_1 h_{1t} - \alpha_2 h_{2t} - \alpha_k h_{kt}$$

Real work:

$$l_t = l_I + l_{II} + l_k = (h_0 - h_I) + (1 - \alpha_1)(h_I - h_2) + (1 - \alpha_1 - \alpha_2)(h_2 - h_k) = h_0 - \alpha_1 h_I - \alpha_2 h_2 - \alpha_k h_k$$

PARAMETERS OF NPP IN SCHEME WITH N-STAGED RHFV

- Effective heat drop of turbine with n^{th} stages:

$$l_i^P = H_i^P = h_0 - \sum_{j=1}^n \alpha_j h_j - \alpha_k h_k$$

- By substituting $\alpha_k = 1 - \sum_{j=1}^n \alpha_j$ we get:

$$H_i^P = h_0 - h_k - \sum_{j=1}^n \alpha_j (h_j - h_k)$$

PARAMETERS OF NPP IN SCHEME WITH N-STAGED RHFV

- Coefficient of turbine power decreasing:

$$y_j = \frac{h_j - h_k}{h_0 - h_k}$$

- Turbine power using values of y_j :

$$H_i^P = h_0 - h_k - \sum_{j=1}^n \alpha_j (h_j - h_k) = H_i \left(1 - \sum_{j=1}^n \alpha_j y_j \right)$$

EFFICIENCY OF NPP IN SCHEME WITH N-STAGED RHFV

■ Efficiency of theoretical NPP with RHFV:

$$\eta_t^p = \frac{H_0^p}{q_1^p} = \frac{H_0^p}{h_0 - h_{fw}} = \frac{\alpha_k (h_0 - h_{kt}) + \sum_{j=1}^n \alpha_j (h_0 - h_{jt})}{h_0 - h_{fw}}$$

■ Efficiency of real NPP with RHFV :

$$\eta_t^p = \frac{H_i^p}{q_1^p} = \frac{H_i^p}{h_0 - h_{fw}} = \frac{\alpha_k (h_0 - h_k) + \sum_{j=1}^n \alpha_j (h_0 - h_j)}{h_0 - h_{fw}}$$

EFFECT OF RHFV ON NPP EFFICIENCY

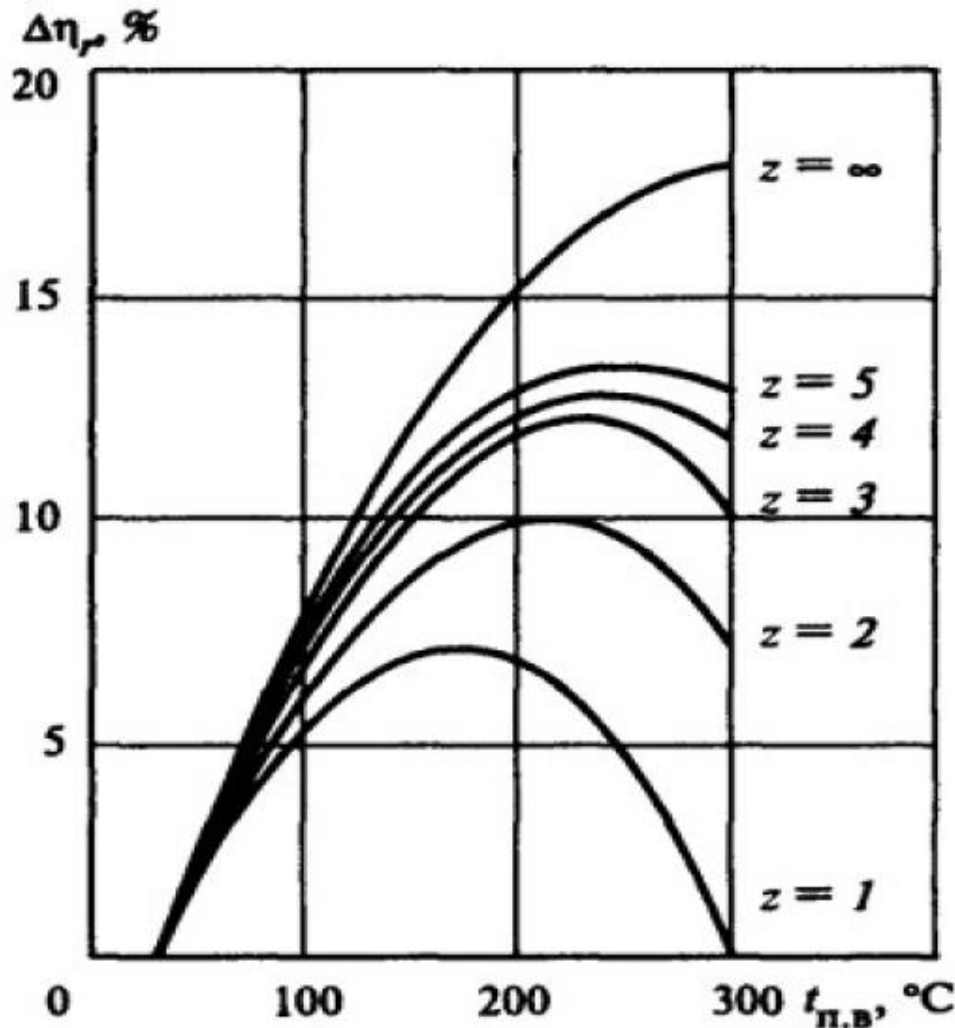
- Energy coefficient with regeneration:

$$A^p = \frac{\sum_{j=1}^n \alpha_j (h_0 - h_j)}{\alpha_k (h_0 - h_k)}$$

- Efficiency of theoretical NPP with RHFV:

$$\eta_i^{reg} = \eta_i^p \frac{1 + A^p}{1 + A^p \cdot \eta_i^p}$$

EFFECT OF NUMBER OF RHFV STAGES ON NPP EFFICIENCY



- Increasing z results into increasing efficiency.
- Maximal efficiency is obtained at optimal temperature of feed water.
- Optimal temperature of feed water increases with z .
- With increasing z the thermal efficiency decreases.

On practice the 7-8 stages of RHFV is applied. It allows to increase efficiency by 15-17 %.

EFFECT OF NUMBER OF RHFWS STAGES ON NPP EFFICIENCY

■ Power of turbine of NPP with RHFWS:

$$N_i = G \cdot H_i^p = G \cdot H_i \cdot \left(1 - \sum_{j=1}^n \alpha_j \cdot y_j \right)$$

■ Steam flow rate on turbine with RHFWS:

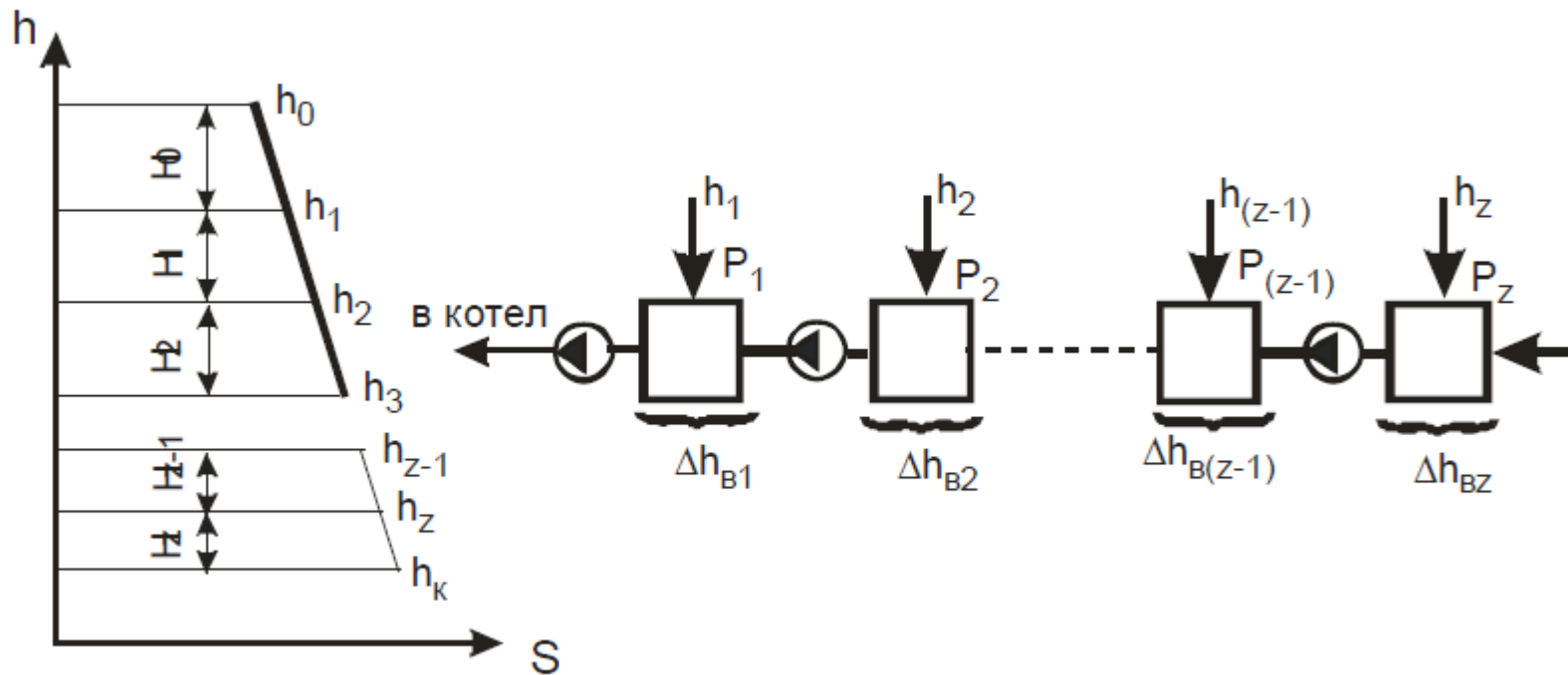
$$G = \frac{N_i}{H_i \cdot \left(1 - \sum_{j=1}^n \alpha_j \cdot y_j \right)}$$

Note. Steam flow rate in NPP with RHFWS is larger in comparison with NPP without regeneration.

$$k_p = \frac{1}{1 - \sum_{j=1}^n \alpha_j \cdot y_j} > 1$$

CHOICE OF NUMBER OF BLEEDS AND PRESSURE

- To reach maximal thermal efficiency of NPP the different methods could be applied.



$$\Delta h_{B1} = H_0; \quad \Delta h_{B2} = H_1; \quad \dots \quad \Delta h_{Bz} = H_{z-1}$$

EVEN DISTRIBUTION OF HEATING

- Assuming the heat taken from 1 kg of steam from one heater to another to be constant the maximal efficiency could be obtained at even distribution of enthalpy increasing.

$$\frac{\Delta h_{B1}}{\Delta h_{B2}} = \dots = \frac{\Delta h_{B(z-1)}}{\Delta h_{Bz}} = m$$

$$\Delta h_{Bn} = \frac{(h'_0 - h'_K)}{z + 1}$$

$$m = \sqrt[z]{\frac{\Delta h_{\Pi 1}}{\Delta h_{\Pi K}}}$$

EVEN DISTRIBUTION OF FEED WATER ENTROPY

$$\Delta S = \frac{(s'_0 - s'_K)}{z + 1}$$

$$\Delta S = \frac{(s'_{\text{ПВ}} - s'_K)}{z}$$

EVEN TEMPERATURE RELATION

$$\frac{T_1}{T_2} = \frac{T_2}{T_3} = \dots\dots\dots = \frac{T_{z-1}}{T_z} = \frac{T_z}{T_K}$$

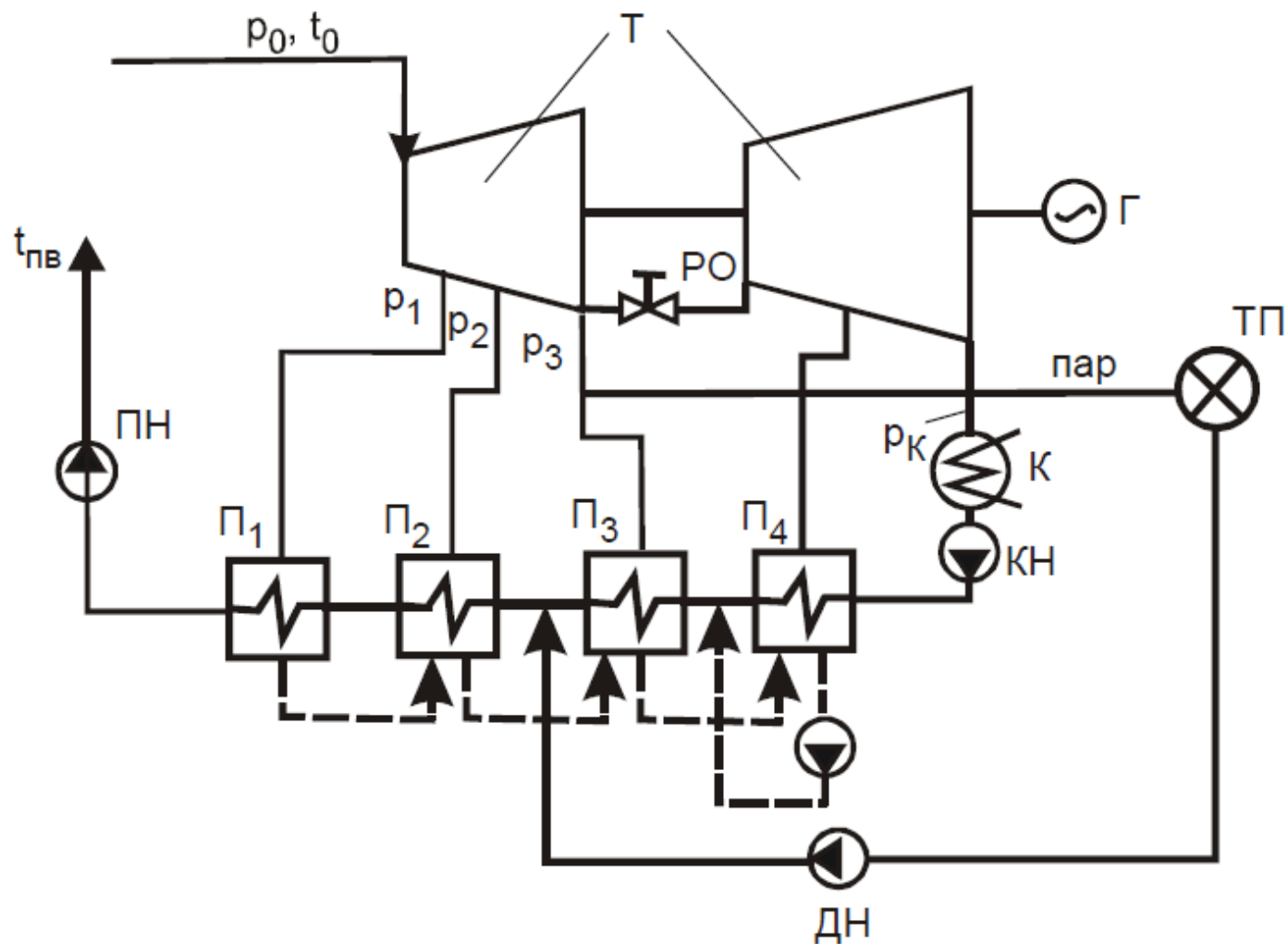
INDIFFERENT POINT

$$\frac{\Delta h_{B1}}{\Delta h_{B2} + \Delta h_{\Pi\Pi}} = \frac{\Delta h_{B2}}{\Delta h_{B3}} = m$$

$$\Delta h_{\Pi\Pi} = (h_{\Pi\Pi 2} - h_{\Pi\Pi 1}) \frac{h_0 - h_{\Pi\Pi 1}}{h_0 - h'_1}$$

$$\Delta h_{B1} = 1,7 \cdot \Delta h_m$$

REGENERATION IN NPP WITH HEAT SUPPLY



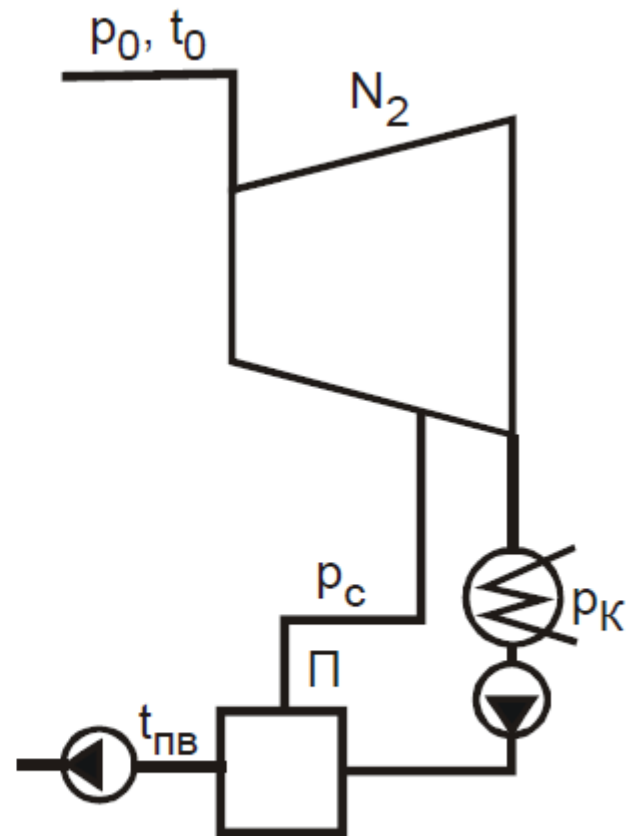
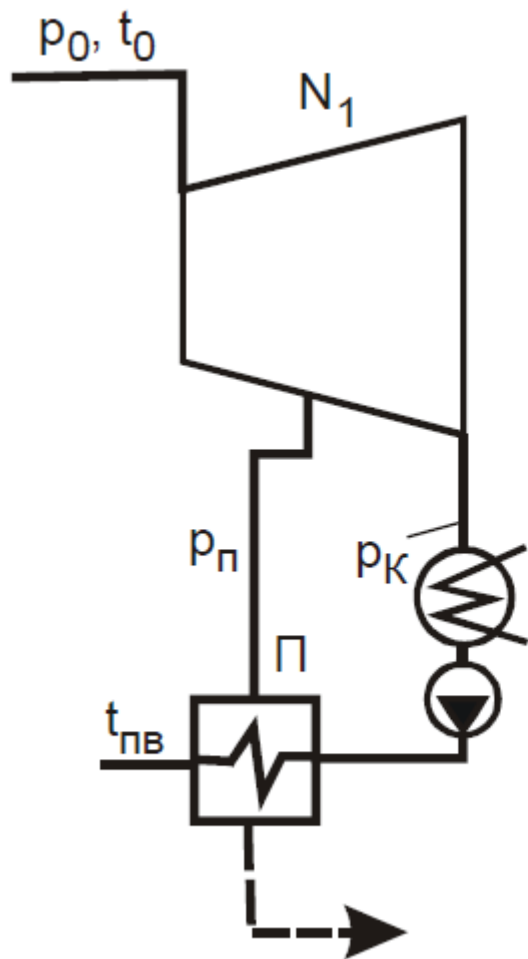
CHOICE OF OPTIMAL TEMPERATURE OF FEED WATER

Choice of optimal feed water temperature is complex technic-economic task.

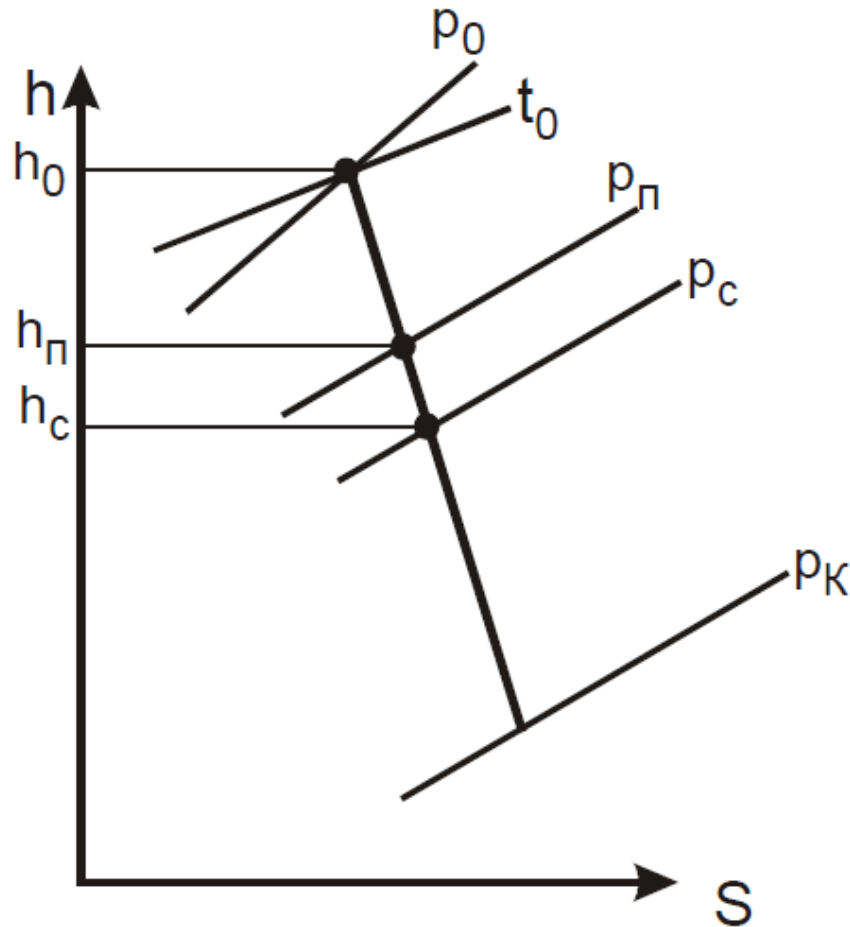
Factors affecting regeneration parameters:

- Specific power of equipment;
 - Initial parameters of steam;
 - Cost of materials and fuel.
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- Taking into account economic factors the amount of regenerative heaters is chosen to be 7-9 in Russia and up to 11 abroad. The technic-economic optimal temperature of feed water is lower than thermodynamic optimal.

COMPARISON OF SURFACE-TYPE AND MIXING HEATERS



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MIXING HEATERS

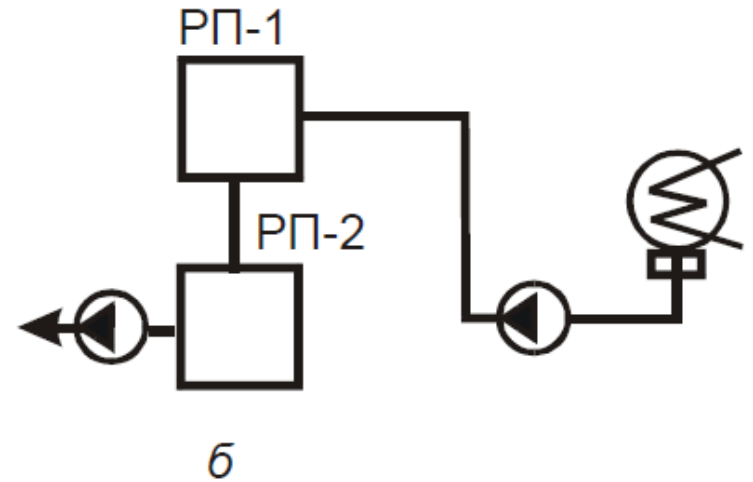
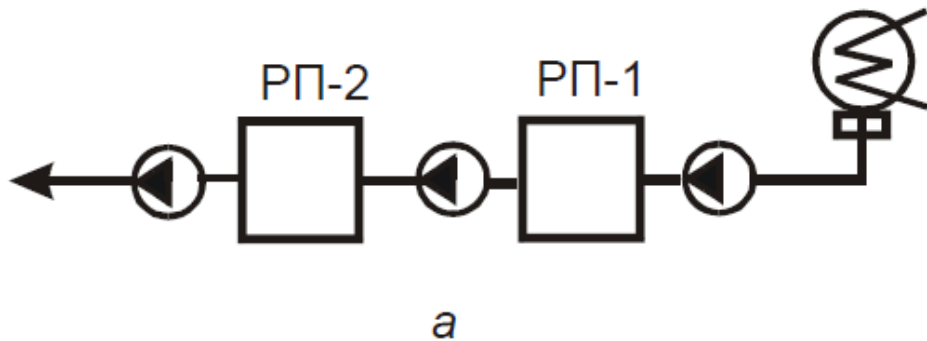
Advantages:

- Simplicity;
- Lower cost;
- Higher reliability;
- Simplicity of exploitation and repairs;
- Lesser concentration of metals and oxides.
- **NO UNDERHEATING.**

Disadvantages:

- Need for additional pumps;
- Need for additional protection from droplets entering turbine (hard to realize at high pressures).

SCHEMES OF MIXING HEATERS



The heaters of mixing type is applied for the first two steps of regenerative heating.

SURFACE HEATERS

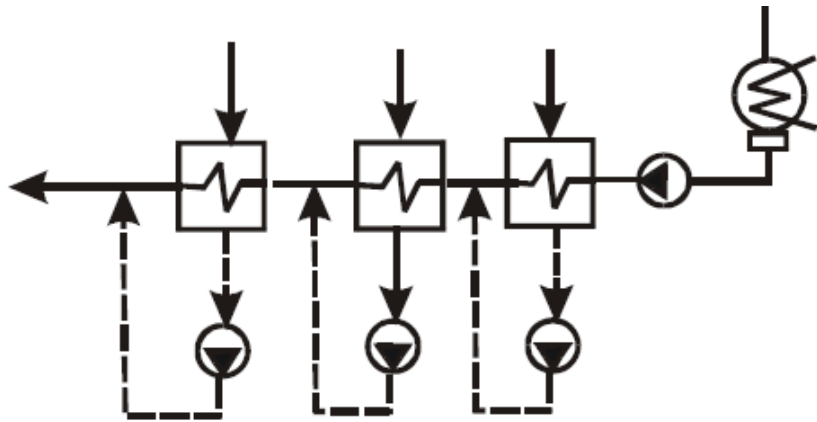
Advantages:

- Independence of steam and water pressures;
- Possibility to apply one pump.

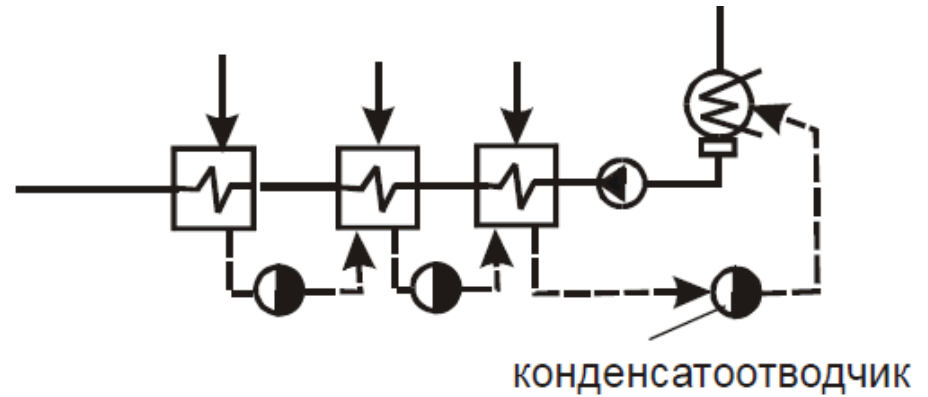
Disadvantages:

- Complexity;
- Lesser reliability;
- Higher costs;
- **UNDERHEATING.**

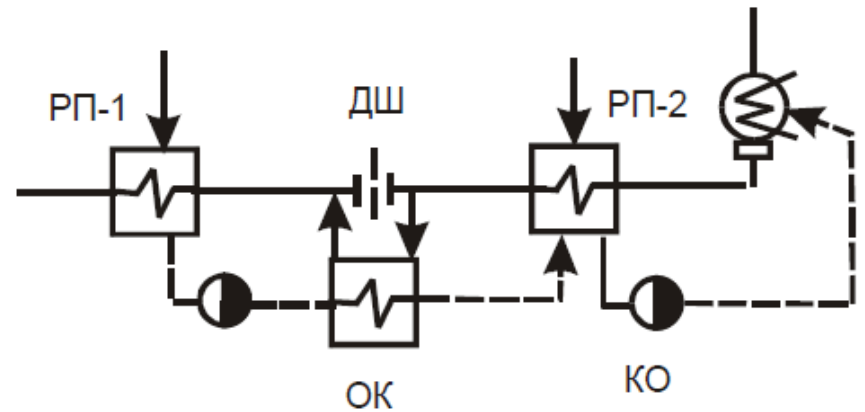
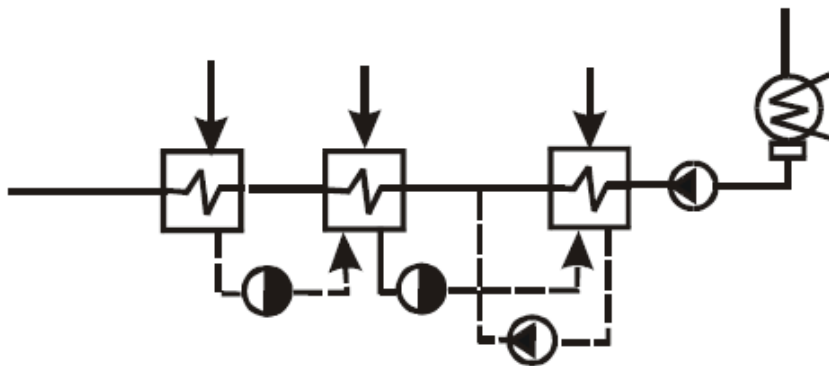
SCHEMES OF SURFACE HEATERS



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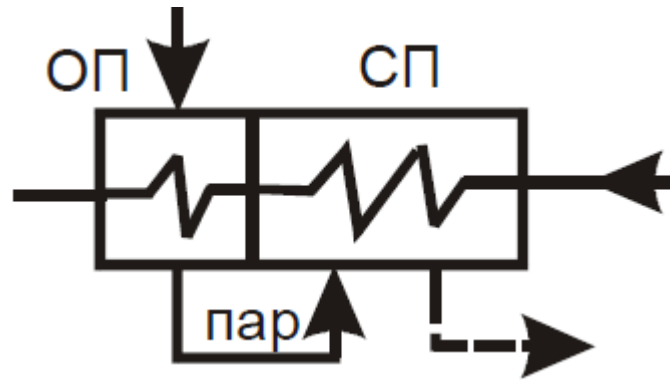
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COMPARISON OF SCHEMES

Scheme	Advantages	Disadvantages
A	Maximal efficiency	Many pumps pumping water with high temperature
B	Simplicity No pumps needed	Steam condensate push out steam from next heaters Additional losses into condenser
C	Intermediate solution between schemes A and B Aim: decrease “pushing out”	
D	Main feature: temperature of condensate in drainage cooler is higher by 10-15 °C in comparison with feed water temperature.	

STEAM COOLER

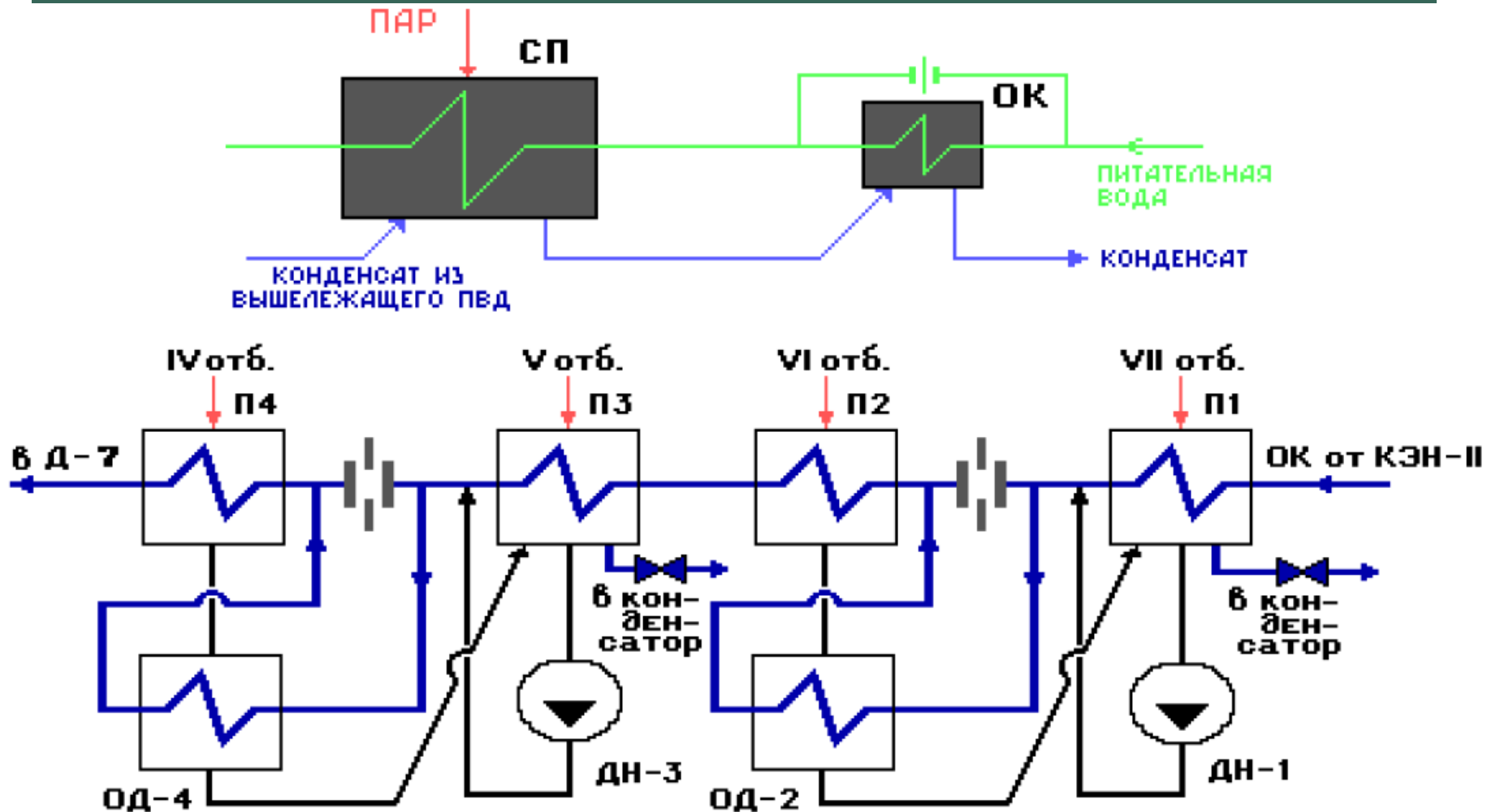


Temperature of steam after steam cooler is higher by 10-15 °C than saturation temperature.

Application of steam cooler gives:

- *higher feed water temperatures on the outlet;*
- *increased efficiency of regeneration;*
- *increased bleed steam consumption.*

DRAINAGE/CONDENSATE COOLER





THANK YOU FOR YOUR ATTENTION