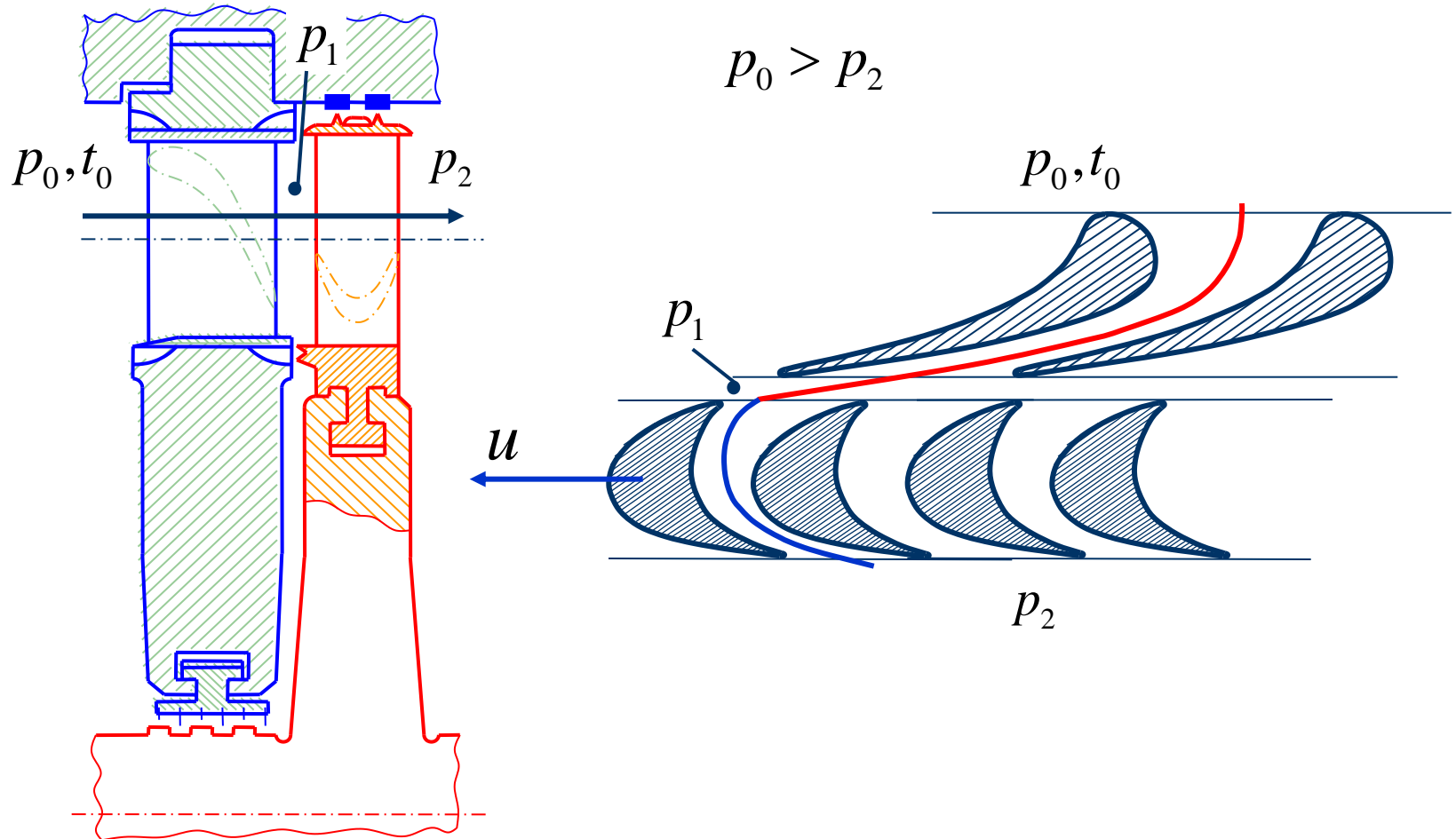


## 2. GAS FLOW IN THE TURBINE STAGE CHANNELS



## 2.1. Basic equations of compressible fluid motion

System of equations (!!!), *adequately* describes gas flow through channels:

- Equation of state
- Continuity equation
- Momentum equation
- Energy-conservation equation

## 2.1.1. Equation of state

The state of matter can be uniquely determined if we know two independent parameters.

**See: Lecture 1**

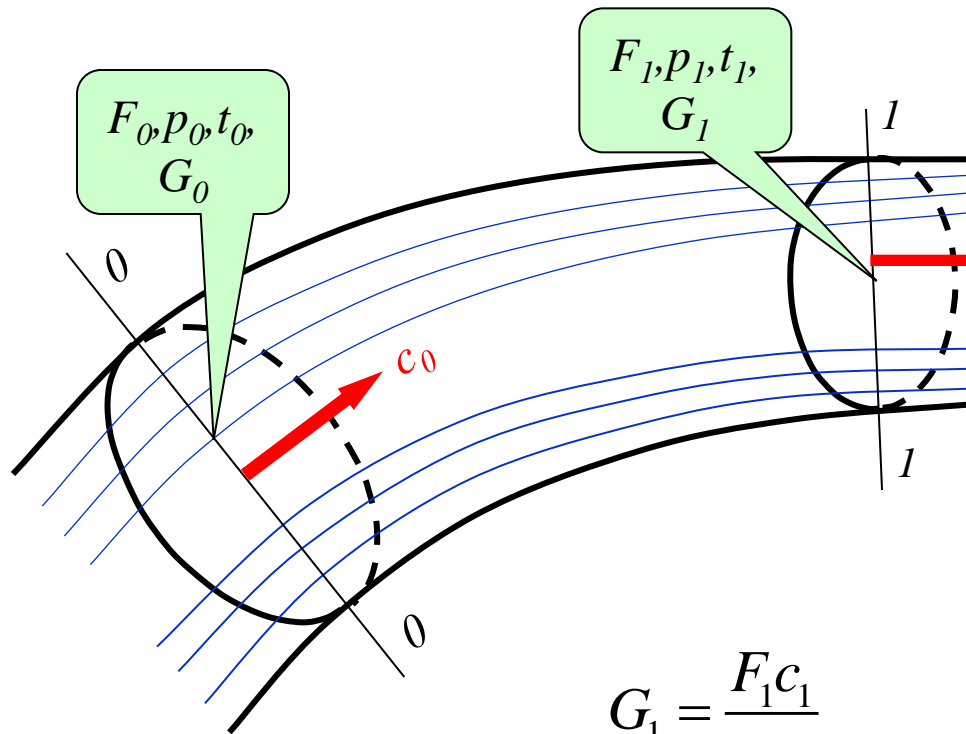
The state changes as a result of the **thermodynamic process**.

We consider the **adiabatic** process - without external heat supply.

An ideal adiabatic (**isentropic**) process is described by the equation:

$$p v_t^k = \text{const}$$

## 2.1.2. Continuity equation



Find the **volume flow rate** [ $m^3/s$ ], passing through section 0-0:

$$V_0 = F_0 c_0 = G_0 v_0$$

$$\frac{m^3}{s} = m^2 \frac{m}{s} = \frac{m^2}{s} \frac{m^3}{m^3}$$

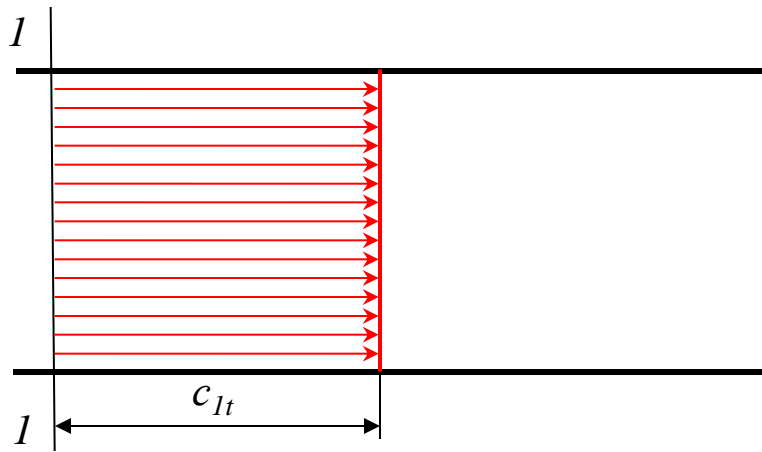
$$v = f(p_0, t_0)$$

$$G_0 = \frac{F_0 c_0}{v_0}$$

$$G_1 = \frac{F_1 c_1}{v_1}$$

$$G_0 = G_1 \quad \frac{F_0 c_0}{v_0} = \frac{F_1 c_1}{v_1}$$

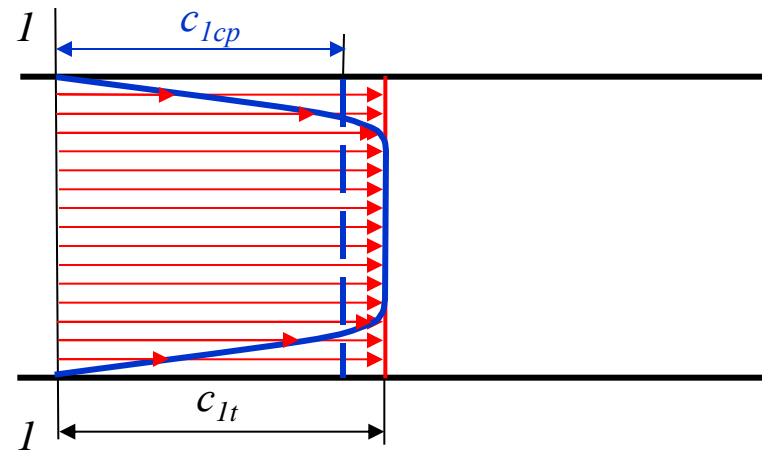
$$G = F \frac{c}{v} = \text{const}$$



a) Ideal flow is the one without friction

$$G_{1t} = \frac{F_1 c_{1t}}{v_{1t}}$$

$$v_{1t} = f(p_1, t_{1t}) = \varphi(p_1, s_0)$$



b) Real flow is the one where viscous forces (friction) are taken into account

$$G_{1\partial} = \int_{(F_1)} \frac{c_1}{v_1} dF_1 = F_1 \frac{c_{1cp}}{v_{1cp}}$$

when  $c_1 = \gamma(F_1)$  and  $v_1 = \varphi(F_1)$

$$v_{1cp} = f(p_1, t_1)$$

$$c_{1cp} = \frac{\int_{(F_1)} c_1 dF_1}{F_1}$$

$$G_1 = F_1 \frac{c_1}{v_1}$$

In the general case:

$$G_i = F_i \frac{c_i}{v_i}$$

Continuity equation in a differential form

$$G = F \frac{c}{v} = \text{const}$$

Take the logarithm of the continuity equation

$$\ln G = \ln F + \ln c - \ln v$$

Differentiate this expression (with  $G = \text{const}$  in mind)

$$0 = \frac{dF}{F} + \frac{dc}{c} - \frac{dv}{v}$$

$$\frac{dF}{F} = \frac{dv}{v} - \frac{dc}{c}$$

A) If  $\frac{dc}{c} > \frac{dv}{v}$ ,  $dF < 0$

The channel is to get narrowed.

B) ) If  $\frac{dc}{c} < \frac{dv}{v}$ ,  $dF > 0$

The channel is to get expanded.

## 2.1.3. Equations of momentum

A) The notion of momentum and energy in dynamic systems

**Newton's second law:**

- for uniformly accelerated motion

$$R = ma$$

where  $R$  is force,  $N$ ;  $m$  is mass,  $kg$ ;  $a$  is acceleration,  $m/s^2$ .

- for motion with variable acceleration (differential form of the equation):

$$R = m \frac{\partial^2 s}{\partial \tau^2}$$

where  $s$  is distance;  $\tau$  is time.



$$\frac{\partial^2 s}{\partial \tau^2} = \frac{d}{d\tau} \left( \frac{ds}{d\tau} \right) = \frac{dc}{d\tau}$$

$$\frac{ds}{d\tau} = c \quad \text{- velocity, m/s.}$$

$$R = m \frac{dc}{d\tau} \xrightarrow[\text{variables}]{\text{divide}} Rd\tau = mdc$$

$$\int_0^\tau Rd\tau = \int_{c_0}^c mdc$$

$$R\tau = mc - mc_0 \quad \text{Equation of momentum}$$

Force impulse is equal to the change in the amount of motion

the amount of motion is changed by the force impulse

$$\frac{\partial^2 s}{\partial \tau^2} = \frac{dc}{d\tau} \frac{ds}{ds} = \frac{ds}{d\tau} \frac{dc}{ds} = c \frac{dc}{ds}$$

$$R = mc \frac{dc}{ds} \longrightarrow Rds = mcdc$$

$$\int_0^s Rds = \int_{c_0}^c mcdc$$

$$Rs = \frac{mc^2}{2} - \frac{mc_0^2}{2}$$

Force multiplied by the distance is mechanical work

Kinetic energy

## B) Applying the momentum equation for liquid (gas) circulation through the channel

### I. Phenomenological approach

If the second mass flow  $G$  [kg/s] and flow velocities at the channel input  $c_1$  [m/s] and output  $c_2$  [m/s] are known, we can determine the force  $R$  [N], which changed the amount of motion.

$$R' = G \left( \vec{c}_2 - \vec{c}_1 \right)$$

As according to **Newton's third law**, to every action there is always opposed and equal reaction, we can determine the force with which the flow acts on the channel walls.

$$R = -R' = G \left( \vec{c}_1 - \vec{c}_2 \right)$$

### II. The problem of determining the actual velocity